



Fictitious domain methods for CFD in the context of nuclear power plants

DE LA RECHERCHE À L'INDUSTRIE

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► Introduction

- Fictitious domain approach
- Industrial context

► Numerical methods

- Immersed Spread Interface
- Jump Embedded Boundary Condition
- Penalized Direct Forcing
- Regularized Direct forcing

► Validation cases

- Thermal-hydraulic
- Navier-Stokes

► Industrial cases

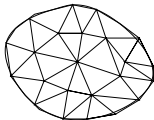
- Steam generator thermal balance
- Nuclear waste vitrification process
- In-vessel flow limiter

► Conclusions and perspectives

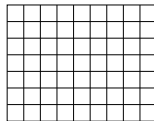
Introduction

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► **Motivations** Computational domain \leftrightarrow Mesh??



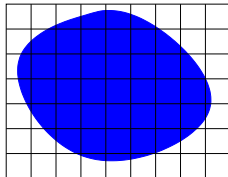
Body-fitted mesh



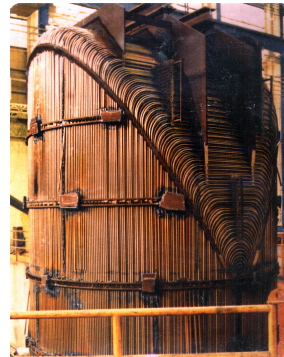
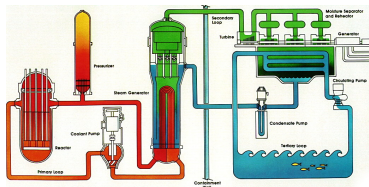
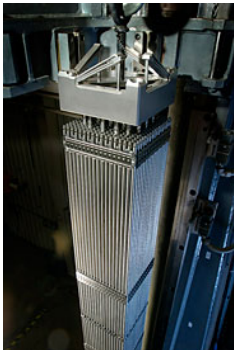
Cartesian mesh

► **Pioneer's work** [Saul'ev, 1963]

- original domain **embedded** in a geometrically bigger and simply-shaped domain : **fictitious domain**
- **computat. domain** = fictitious domain
- **Cartesian mesh** \Rightarrow fast linear solvers, good numerical properties, easy multiresolution techniques

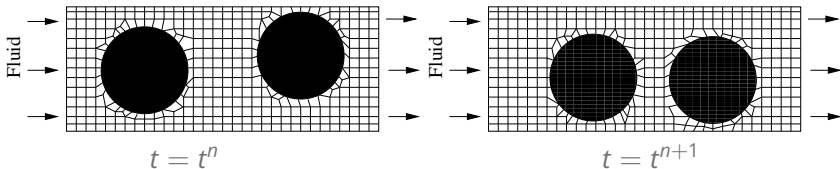


- ▶ Method with a good accuracy/cost ratio :
 - unique formalism for **any immersed B.C.** (Dirichlet, Robin, Neumann) or **mixed** B.C. types
 - moving free surfaces
 - a **unique Cartesian mesh**, no extra unknown, no numerical scheme modification ⇒ **first or second order** method
 - easy AMR implementation
- ▶ Two kinds of immersed boundary approximation :
 - **Spread** interface
 - **Thin** interface
- ▶ Milestones :
 - Scalar convection-diffusion problem
 - Navier-Stokes problem
 - Fluid-structure problem

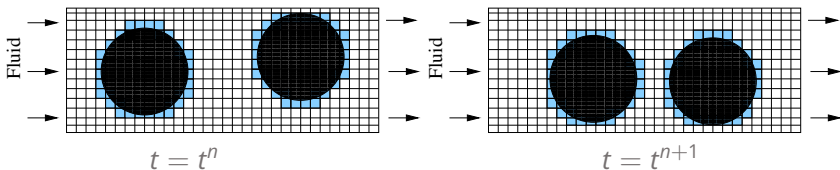


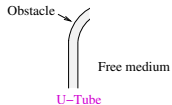
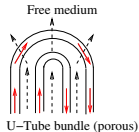
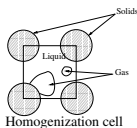
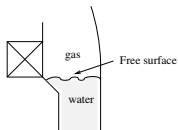
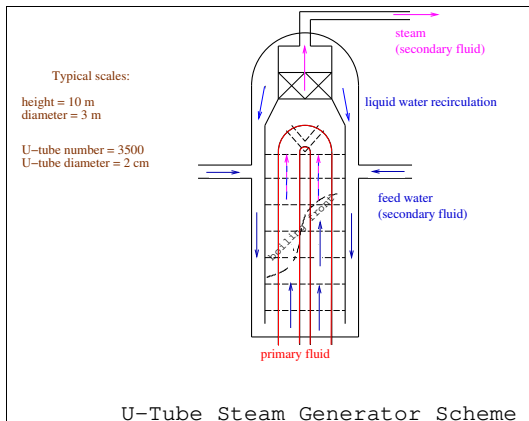
Flow-induced **vibrations** → possible fuel-rod or U-tube **damages**
Fast FSI computations with **large deformations** → Immers. **Bound. Methods**

► With re-meshing....

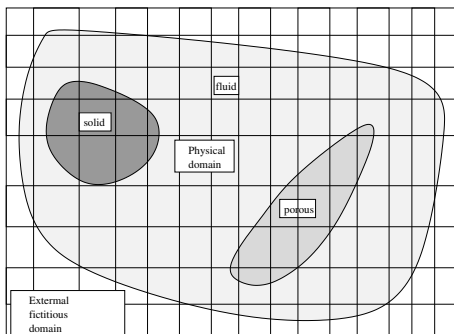


► Without re-meshing....





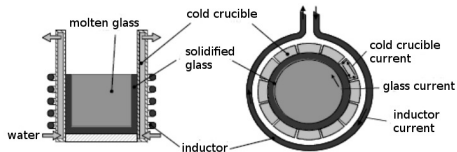
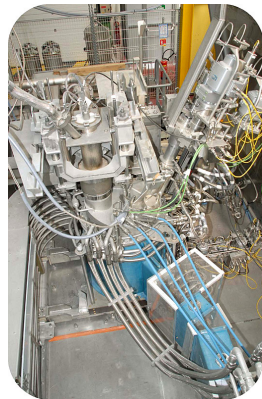
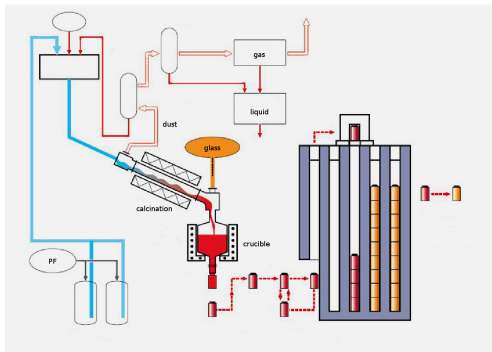
- An (abstract?) SG model



- $$\partial_t(\rho \vec{v}) + \vec{\nabla} \cdot (\rho \vec{v} \otimes \vec{v}) = \vec{f}_v - \vec{\nabla} P + \vec{\nabla} (\mu_T (\vec{\nabla} \vec{v} + \vec{\nabla}^t \vec{v})) - \frac{\mu}{K} \vec{v}$$

[Khadra et al., 2000]

A **unique formalism** for free, porous or blind, fixed or moving regions

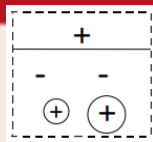


- ▶ Problems involving complex flow regimes combined with time-varying geometries
 - ★ Multiphase multicomponent flow at high temperature (gas bubbles, molten glass)
 - ★ Static (vessel structure, measuring apparatus), moving (mechanical stirrer) bodies

- ▶ From the industrial point of view, the numerical constraints are
 - To develop robust, efficient and easily parallelizable numerical tools
 - use non-boundary conforming methods (several proposed in literature)
 - To get accurate solutions at lower computational cost
 - use high-order methods (\sim second order in nuclear safety context)

A 1-fluid / Front Tracking Model

- ▶ Lagrangian meshes for the bubbles
- ▶ Eulerian phase indicator $\zeta \rightarrow \rho$ and μ

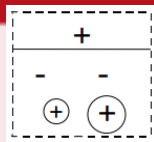


$$\rho = \rho_0 + \zeta(\rho_1 - \rho_0) \quad \mu = \mu_0 + \zeta(\mu_1 - \mu_0)$$

$$-\nabla(P + \rho g \mathbf{z}) + (\rho_1 - \rho_0) g \mathbf{z} \cdot \nabla \zeta$$

A 1-fluid / Front Tracking Model

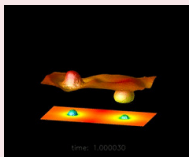
- ▶ Lagrangian meshes for the bubbles
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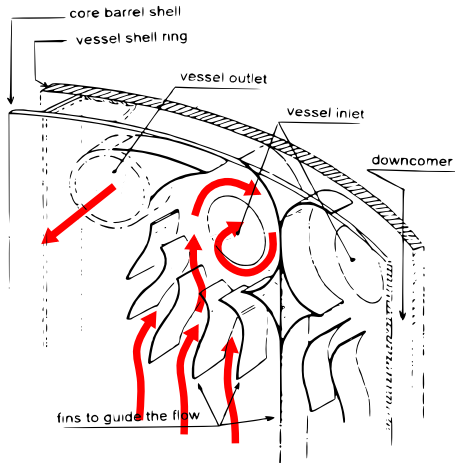


$$\rho = \rho_0 + \zeta(\rho_1 - \rho_0) \quad \mu = \mu_0 + \zeta(\mu_1 - \mu_0)$$

$$-\nabla(P + \rho g \mathbf{z}) + (\rho_1 - \rho_0) g \mathbf{z} \cdot \nabla \zeta$$

- ▶ Interfaces move with the continuous-phase velocities : $d_t \mathbf{X}_L = \mathcal{I}(\mathbf{u}, \mathbf{X}_L)$
- ▶ Lagrangian remeshing process and fragmentation/collision models





Patented Passive Safety System

- ▶ Located in a PWR core
- ▶ Composed of thin fins which :
 - Nominal flow not disturbed
 - Vortex in reverse flow
 ⇒ *Postpone core dewatering*

Preliminary studies

- ▶ At a system scale
 - CATHARE
 - ⇒ *Operating limits*
- ▶ At a component scale
 - GENEPI2
 - ↔ *Upscaling*
 - ⇒ *Demonstrated interest*

Problem description

- ▶ Thermal-hydraulics
- ↔ *Two-phase flow*
- ↔ *Compressible*
- ↔ *Low Mac Number*
- ▶ Passive Safety systems
- ↔ *Design and shape optimization*
- ⇒ *Large number of simulations*
- ⇒ *Need of a fast tool*



Modeling choices

- ▶ Homogeneous Equilibrium Model
- ▶ Projection scheme
- ▶ “Body-fitted” approach
- ▶ “Immersed boundary” approach

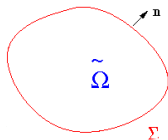
Numerical methods

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- Original problem ($\tilde{\mathcal{P}}$) in $\tilde{\Omega} \subset \mathbb{R}^d$:

For $\tilde{\mathbf{a}} \in (L^\infty(\tilde{\Omega}))^{d \times d}$, $\tilde{\mathbf{v}} \in (L^\infty(\tilde{\Omega}))^d$, $\tilde{b} \in L^\infty(\tilde{\Omega})$ and $\tilde{f} \in L^2(\tilde{\Omega})$, find $\tilde{u} \in H^1(\tilde{\Omega})$ solution of :

$$(\tilde{\mathcal{P}}) : -\operatorname{div}(\tilde{\mathbf{a}} \cdot \nabla \tilde{u}) + \operatorname{div}(\tilde{\mathbf{v}} \tilde{u}) + \tilde{b} \tilde{u} = \tilde{f} \quad \text{in } \tilde{\Omega}$$



with the B.C. on $\partial\tilde{\Omega}$

$$\left\{ \begin{array}{l} \bullet \text{ Dirichlet : } \tilde{u} = u_D, u_D \in H^{1/2}(\partial\tilde{\Omega}) \\ \bullet \text{ Robin (or Neumann) : } -(\tilde{\mathbf{a}} \cdot \nabla \tilde{u}) \cdot \mathbf{n} = \alpha_R (\tilde{u} - u_R) + g_R, \\ \quad 0 \leq \alpha_R \in L^\infty(\partial\tilde{\Omega}), \text{ and } g_R \in L^2(\partial\tilde{\Omega}) \end{array} \right.$$

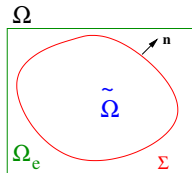
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- Fictitious Domain approach : $\Omega = \tilde{\Omega} \cup \Sigma \cup \Omega_e$

- Which fictitious problem (\mathcal{P}) solved in Ω ?
- Which B.C. on Σ , coefficients on the external domain Ω_e and B.C. on $\partial\Omega$?

- Fictitious problem (\mathcal{P}) solved in Ω_h :

Find $u_\eta^h \in H^1(\Omega_h)$ such that

$$(\mathcal{P}) \begin{cases} \operatorname{div}(-\mathbf{a}\nabla u_\eta^h + \mathbf{v}u_\eta^h) + b u_\eta^h = f & \text{in } \Omega_h \\ (\tilde{\mathcal{P}})'s \text{ B.C.} & \text{on } \tilde{\Gamma}_h = \partial\Omega_h \cap \partial\tilde{\Omega}_h \\ \text{suitable B.C. for } u_\eta^h & \text{on } \Gamma_{e,h} = \partial\Omega_h \setminus \partial\tilde{\Omega}_h \end{cases}$$

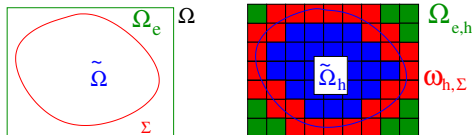
with $\mathbf{a} \in (L^\infty(\Omega_h))^{d \times d}$, $\mathbf{v} \in (L^\infty(\Omega_h))^d$, $b \in L^\infty(\Omega_h)$ and $f \in L^2(\Omega_h)$, such that $\mathbf{a}|_{\tilde{\Omega}_h} = \tilde{\mathbf{a}}|_{\tilde{\Omega}_h}$, $\mathbf{v}|_{\tilde{\Omega}_h} = \tilde{\mathbf{v}}|_{\tilde{\Omega}_h}$, $b|_{\tilde{\Omega}_h} = \tilde{b}|_{\tilde{\Omega}_h}$, $f|_{\tilde{\Omega}_h} = \tilde{f}|_{\tilde{\Omega}_h}$

$$\mathbf{a}|_{\Omega_{e,h}} = \mathbf{Id} \text{ or } \ll \mathbf{Id}, \mathbf{v}|_{\Omega_{e,h}} = \mathbf{0}, b|_{\Omega_{e,h}} = 0, f|_{\Omega_{e,h}} = 0$$

- Robin B.C. : $-(\tilde{\mathbf{a}} \cdot \nabla \tilde{u}) \cdot \mathbf{n}|_\Sigma = \alpha_R (\tilde{u}|_\Sigma - u_R) + g_R$
- Dirichlet B.C. : $\tilde{u}|_\Sigma = u_D$ by L^2 penalization of Robin B.C. :

$$\alpha_R = \frac{1}{\eta}, \alpha_R \xrightarrow{\eta \rightarrow 0} +\infty, u_R = u_D, g_R = 0$$

- Spread Approximation $\omega_{h,\Sigma}$ of Σ :



- Robin B.C. : $-(\tilde{\mathbf{a}} \cdot \nabla \tilde{u}) \cdot \mathbf{n}|_{\Sigma} = \alpha_R (\tilde{u}|_{\Sigma} - u_R) + g_R$
Transmission problem between $\tilde{\Omega}$ and Ω_e :

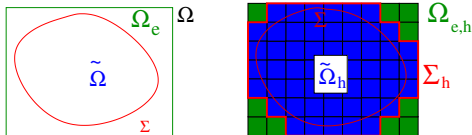
$$\psi = \mathbf{a}, \mathbf{v}, \dots \quad \psi = \begin{cases} \tilde{\psi} & \text{in } \tilde{\Omega} \\ \psi_e & \text{in } \Omega_e \end{cases} \quad \llbracket \psi \rrbracket_{\Sigma} = \psi^+ - \psi^-$$

$$-\operatorname{div}(\mathbf{a} \cdot \nabla u) + \operatorname{div}(\mathbf{v} u) + u = f - \llbracket (\mathbf{a} \cdot \nabla u) \cdot \mathbf{n} \rrbracket_{\Sigma} \delta_{\Sigma} + \llbracket (\mathbf{v} \cdot \mathbf{n}) \rrbracket_{\Sigma} u \delta_{\Sigma}$$

$$-(\mathbf{a} \cdot \nabla u) \cdot \mathbf{n}|_{\Sigma}^- = \alpha_R (u|_{\Sigma} - u_R) + g_R \quad \text{and} \quad -(\mathbf{a} \cdot \nabla u) \cdot \mathbf{n}|_{\Sigma}^+ \simeq 0 \quad \text{with } \mathbf{a}|_{\Omega_e} = \eta \mathbf{Id}$$

Ad-hoc coefficient ϵ_h to approximate δ_{Σ} on $\omega_{h,\Sigma}$

- ▶ Thin Approximation Σ_h of Σ :



- ▶ Model of flux and solution jumps [Angot, 2003]

$$\begin{cases} \overline{[(\mathbf{a} \nabla u_\eta^h) \cdot \mathbf{n}]_{\Sigma_h}} = \alpha \overline{u_\eta^h}|_{\Sigma_h} - q & \text{on } \Sigma_h \\ (\mathbf{a} \nabla u_\eta^h) \cdot \mathbf{n}|_{\Sigma_h} = \beta [u_\eta^h]_{\Sigma_h} - g & \text{on } \Sigma_h \end{cases}$$

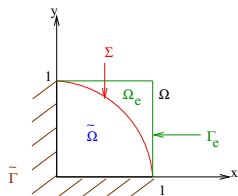
- ▶ Immersed transmission conditions

No exterior control : $\beta = \frac{\alpha}{4}$

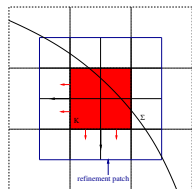
$$\begin{cases} \psi_\Sigma^- := -(\mathbf{a} \cdot \nabla u)^- \cdot \mathbf{n}|_{\Sigma_h} = \frac{\alpha}{2} u^-|_{\Sigma_h} + g - \frac{q}{2} \\ \psi_\Sigma^- = \alpha_R (u^-|_{\Sigma_h} - u_R) + g_R \end{cases}$$

- ▶ Ad-hoc coefficient ϵ_h to approximate $\int_\Sigma \psi_\Sigma^- dS$ by $\int_{\Sigma_h} \psi_{\Sigma_h}^- \frac{1}{\epsilon_h} dS$

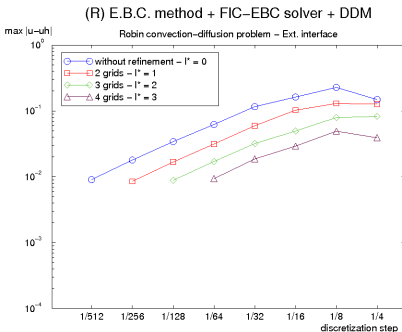
Convergence with the space step (h) - $\eta = 10^{-12}$



(a) Computational domain



(b) A patch of the FIC-EBC method



(c) Robin test case

\Rightarrow Methods in $\mathcal{O}(h_f)$ with h_f : space step of the **finest** local refinement grid.

Balance equations on an open set Ω (dilatible Navier-Stokes)

$$\left\{ \begin{array}{ll} \partial_t(\rho \mathbf{u}) + \nabla \cdot (\rho \mathbf{u} \otimes \mathbf{u} - \bar{\bar{\sigma}} + \bar{\bar{l}}p) = \rho \mathbf{g} & \text{on } \Omega \\ \nabla \cdot (\rho \mathbf{u}) = 0 & \text{on } \Omega \\ +\text{B.C.} & \text{on } \partial\Omega \\ +\text{I.C.} & \text{on } \Omega \end{array} \right.$$

- ρ , fluid density
- p , fluid pressure
- \mathbf{g} , gravity vector
- \mathbf{u} , fluid velocity
- $\bar{\bar{l}}$, identity tensor of $M_3(\mathbb{R})$
- $\bar{\bar{\sigma}}$, deviatoric stress tensor

Projection scheme to solve Navier-Stokes

$$\frac{\rho^n}{\delta t} (\mathbf{u}^* - \mathbf{u}^n) + \nabla \cdot \bar{\bar{\Sigma}}^{n,*} = \rho^n \mathbf{g}$$

$$\frac{\rho^n}{\delta t} (\mathbf{u}^{n+1} - \mathbf{u}^*) + \nabla \phi^{n+1} = \mathbf{0}$$

- x^i , approximation of x at time step number $i \in \mathbb{N}$
- δt , time step
- ρ^n , value of density computed a from tabulated equation of state
- $\phi^{n+1} = p^{n+1} - p^n$, pressure corrector
- \mathbf{u}^* , predicted velocity
- $\bar{\bar{\Sigma}}^{n,*} = \rho^n \mathbf{u}^n \otimes \mathbf{u}^* - \bar{\sigma}^* + \bar{l} p^n$

Forcing term addition

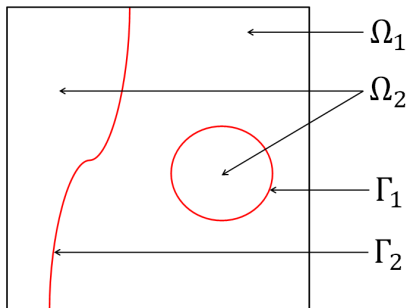
$$\mathbf{f}^{n+1} = \frac{\chi}{\eta \delta t} \rho^n (\mathbf{u}_\Gamma - \mathbf{u}^{n+1})$$

- $\eta \ll 1$, penalty parameter
- χ , characteristic function of Γ
- \mathbf{u}_Γ , imposed velocity on Γ

Adaption to projection scheme

$$\mathbf{f}_P^{n+1} := \frac{\chi}{\eta \delta t} \rho^n (\mathbf{u}_\Gamma - \mathbf{u}^*)$$

$$\mathbf{f}_C^{n+1} := \frac{\chi}{\eta \delta t} \rho^n (\mathbf{u}^* - \mathbf{u}^{n+1})$$



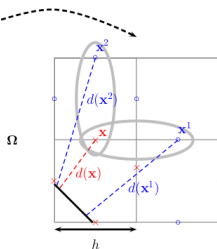
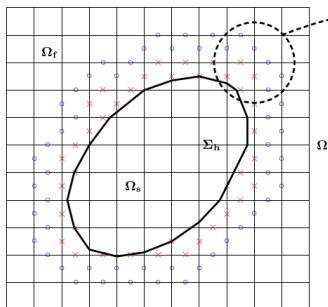
$\Omega = \Omega_1 \cup \Omega_2$, "fluid" domain

$\Gamma = \Gamma_1 \cup \Gamma_2$, immersed boundary

⇒ A robust linear interpolation scheme (second-order)

$$\mathbf{u}_i(\mathbf{x}) := \mathbf{u}_s(\Pi_{\Sigma}(\mathbf{x})) + \frac{d(\mathbf{x})}{N} \sum_{\rho=1}^N \frac{\mathbf{u}(\mathbf{x}^{\rho}) - \mathbf{u}_s(\Pi_{\Sigma}(\mathbf{x}^{\rho}))}{d(\mathbf{x}^{\rho})} + \mathcal{O}(h^2) \text{ for } \mathbf{x} \in \Omega \text{ s. t. } \chi_s(\mathbf{x}) > 0, d(\mathbf{x}) > 0$$

- Does not rely on a preferred direction as often done in literature
- Based on an averaged reconstruction of velocity gradients near Σ_h
 - ↪ the local influence of fluid flow around \mathbf{x} is fully taken into account



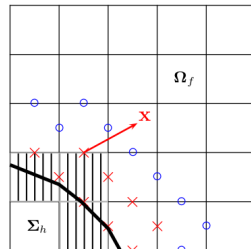
- × forcing or penalized nodes
- associated fluid nodes

- Based on an approximate projection operator (Π_{Σ}) onto Σ_h

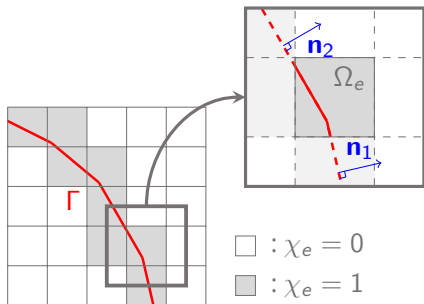
- Π_Σ is defined throughout an algorithm based on this local minimization problem

$$\text{Find } \mathbf{z} (= \Pi_\Sigma(\mathbf{x})) \in \mathcal{V}^{\Sigma_h} \text{ s. t. } J(\mathbf{z}) = \inf_{\mathbf{y} \in \mathcal{V}^{\Sigma_h}} J(\mathbf{y}) \text{ with } J(\mathbf{y}) = \|\mathbf{y} - \mathbf{x}\|^2$$

- \mathcal{V}^{Σ_h} is an immediate neighborhood of Σ_h
 - ★ Σ_h is partially (and locally) reconstructed by the Lagrangian facets contained in the neighboring cells and the “neighbors of neighbors” around \mathbf{x}
- \mathcal{V}^{Σ_h} is defined by a set of plans passing through each Lagrangian facets of Σ_h
- \mathcal{V}^{Σ_h} tends to Σ_h when the space step h decreases



- Resolution of the minimization problem by means of an Uzawa algorithm



Discrete obstacle

- ▶ Characteristic χ_e by element
- ▶ Normal vector by element

Finite Element Formulation

- ▶ Pair of elements (Q_1, Q_0)
- ⇒ *Few degrees of freedom, fast*
- ⇒ *Instabilities soften by diffusion*
- ▶ Lumped mass matrix
- ⇒ *Enhance robustness*

Motivations

- ▶ Interpolation of \mathbf{u}_Γ
- ⇒ *Use of Q_1 basis*
- ▶ House code of the CEA
- ⇒ *Facilitates developments*

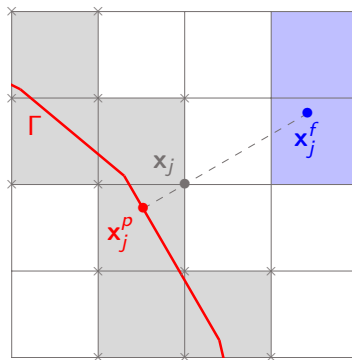
Interpolation of the imposed \mathbf{u}_Γ

- ▶ Projection \mathbf{x}_j^p of a node \mathbf{x}_j on Γ
- ▶ Point "purely fluid" \mathbf{x}_j^f

$$\mathbf{u}_\Gamma = \mathbf{u}_j^p + \frac{\mathbf{u}_j^f - \mathbf{u}_j^p}{\|\mathbf{x}_j^f - \mathbf{x}_j^p\|} \|\mathbf{x}_j - \mathbf{x}_j^p\|$$

"Purely fluid" element

- ▶ No immersed Dirichlet node
i.e. *No node influenced by Γ*

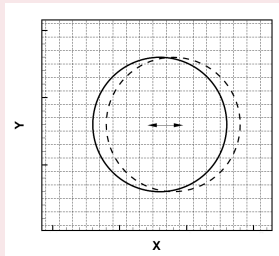


: $\chi_e = 0$, "Purely fluid"

Spurious Force Oscillations ...

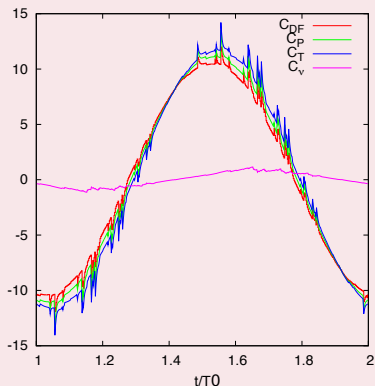
$$x_c(t) = x_c(0) + X_0(1 - \cos(2\pi f_0 t))$$

$$y_c(t) = y_c(0)$$



► Drag coefficient $C_T = C_P + C_V$

$$C_{DF} = \frac{\iint_{\Omega_S} \rho E_{DF} \cdot e_x d\Omega}{E_{ref} \cdot e_x}$$



$$\Delta x_0 = D/16 \text{ and } \Delta t_0 = 0.002 T_0$$

Regularization of \underline{u}_i

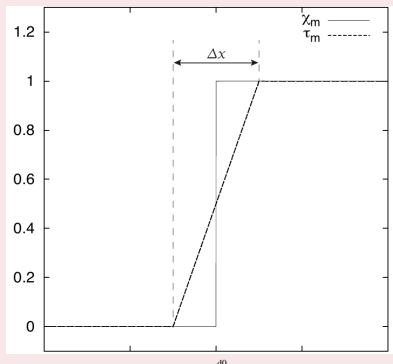
$$\begin{cases} \underline{u}_i &= \xi_m \underline{u}_m + (1 - \xi_m) \underline{u}^* \\ \underline{F}_{RG} &= \frac{\underline{u}_i - \underline{u}^*}{\Delta t} \end{cases}$$

$0 \leq \xi_m(\underline{x}) \leq 1$: regular function of $d(\underline{x})$

$$\underline{F}_{RG} = \xi_m \frac{\underline{u}_m - \underline{u}^*}{\Delta t}$$

An example ...

$$\xi_m(\underline{x}) = \frac{\Delta x + 2[d(\underline{x}) - d_0^m]}{2\Delta x} \equiv \tau_m(\underline{x})$$



Regularization of \underline{u}_j

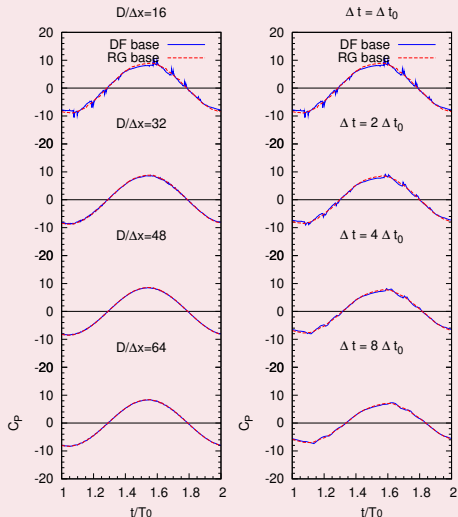
$$\begin{cases} \underline{u}_j & = \xi_m \underline{u}_m + (1 - \xi_m) \underline{u}^* \\ \underline{F}_{RG} & = \frac{\underline{u}_j - \underline{u}^*}{\Delta t} \end{cases}$$

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An example ...

$$\xi_m(\underline{x}) = \frac{\Delta x + 2[d(\underline{x}) - d_0^m]}{2\Delta x} \equiv \tau_m(\underline{x})$$



Validation cases

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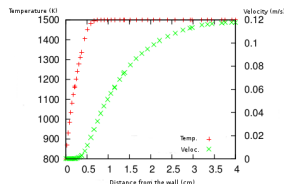
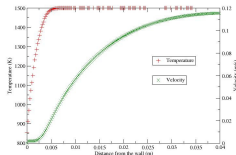
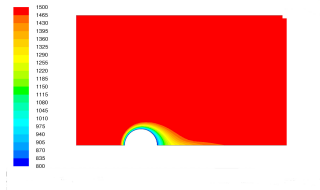
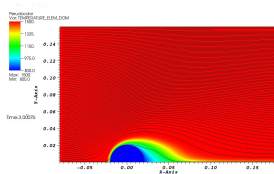
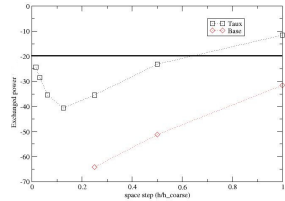
2D molten glass flow over a cold cylinder

- ▶ $\mu(T) \in [1; 10^5]$, $\Lambda(T)$ and Boussinesq approx.
- ▶ Fluent benchmark simulation
(boundary layer : AMR; 14,000 faces)
- ▶ Mesh convergence study (2,400 \rightarrow 10 M cells)

$$\begin{aligned} \partial_t T &= \nabla \left(\frac{\lambda(T)}{\rho_0 C_{P0}} \vec{\nabla} T \right) \\ &+ (1 - \chi_s) \frac{Q}{\rho_0 C_{P0}} \\ &- (1 - \chi_s) \vec{V} \cdot \vec{\nabla} T + \frac{\chi_s}{\eta} (T_i - T) \end{aligned}$$

2D molten glass flow over a cold cylinder

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Penalty parameter η convergence study

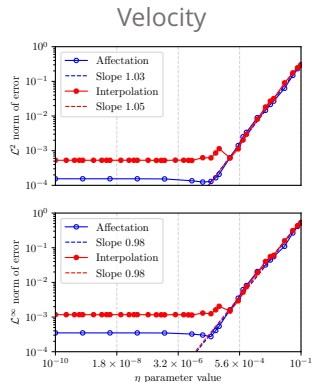
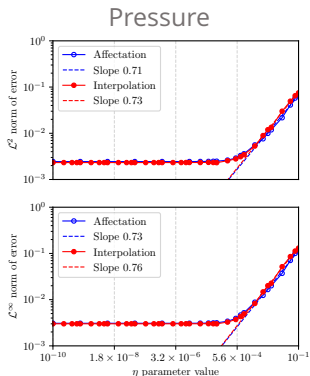
- ▶ Laminar 2D Poiseuille flow (channel aligned with the grid)

Mesh convergence studies

- ▶ Laminar 2D Poiseuille flow (channel tilted with respect to the grid)
- ▶ Taylor-Couette flow
- ▶ Laminar flow around a static circular cylinder
- ▶ Laminar flow around a rotating circular cylinder

Global quantities studies

- ▶ Laminar flow around a static circular cylinder
- ▶ Laminar flow around a rotating circular cylinder
- ▶ Steady laminar flow past a NACA0012 airfoil

Penalty parameter η convergence study (aligned Poiseuille)

↪ *Order of convergence*

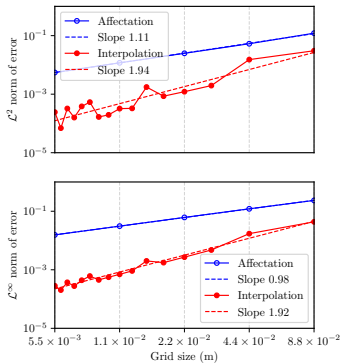
- Consistent with the literature

↪ *Linear interpolation*

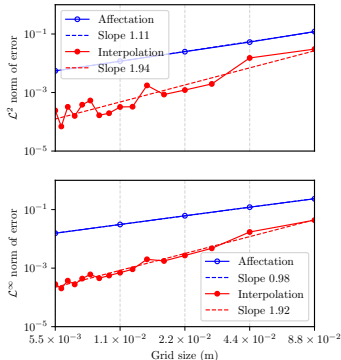
- No degradation

Mesh convergence study (45° tilt Poiseuille)

Velocity along x axis



Velocity along y axis



↪ *Order of convergence*
 — Consistent with literature

↪ *Linear interpolation*
 — Enhances convergence

Aerodynamic coefficients for the flow around a rotating circular cylinder (Re=100)

$$2r/h = 40$$

References in
Literature

$$\overline{C_d} \quad 1.102$$

[1.098-1.189]

$$C_d' \quad 0.091$$

[0.099-1.120]

$$\overline{C_l} \quad 2.562$$

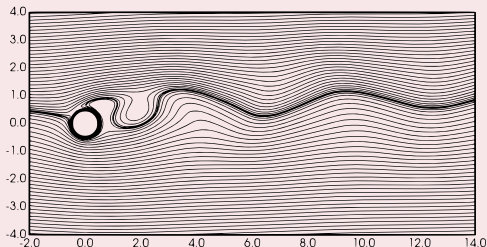
[2.405-2.510]

$$C_l' \quad 0.301$$

[0.360-0.443]

$$St \quad 0.166$$

[0.165-0.173]



Industrial cases

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► The energy balance equation

$$\begin{aligned} \beta \rho \partial_t H + \beta \mathbf{G} \cdot \nabla H - \operatorname{div}(\beta \chi_T \nabla H) \\ = \beta Q - \operatorname{div}(\beta x(H) (1 - x(H)) \rho \mathcal{L} \mathbf{V}_R), \end{aligned}$$

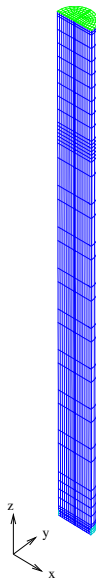
► $\mathbf{G} = \rho \mathbf{V}$, $x(H, P) = \frac{H - H_{lS}}{\mathcal{L}}$ static quality, \mathbf{V}_R relative velocity (drift flux)

► $\chi_T = a |\mathbf{G}| L$: Schlichting's turbulent diffusion coefficient

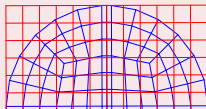
► B.C.

- $H_D = 119,3 \text{ KJ.kg}^{-1}$ (hot leg), $H_D = 118,5 \text{ KJ.kg}^{-1}$ (cold leg),
- $\nabla H \cdot \mathbf{n} = 0$ (wall and out-flow)

► Neptune/pyGene code (F.E. method)



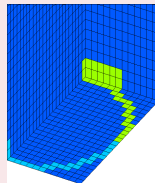
The ISI formulation



$$\begin{aligned}
 & \sum_e \rho_e \int_{\omega_e} \xi^i \beta \partial_t H + \sum_e \mathbf{G}_e \cdot \int_{\omega_e} \xi^i \beta \nabla H \\
 & + \sum_e \chi_{Te} \int_{\omega_e} \beta \nabla H \cdot \nabla \xi^i - \sum_e \chi_{Te} \int_{\partial \omega_e \cap \partial \Omega_h} \xi^i \beta \nabla H \cdot \mathbf{n} + \sum_e (\alpha_R)_e \sum_j H_j \int_{\omega_e} \xi^i \xi^j \\
 & = \sum_e Q_e \int_{\omega_e} \xi^i \beta - \sum_e \int_{\omega_e} \xi^i \operatorname{div}(\beta x(1-x) \rho \mathbf{L} \mathbf{V}_R) + \sum_e [(\alpha_R)_e (H_D)_e + \frac{(g_R)_e}{\epsilon_e}] \int_{\omega_e} \xi^i
 \end{aligned}$$

► Immersed B. C. :

- In-flow, Dirichlet : L^2 penalization $\alpha_R = \frac{1}{\eta}$
 $0 < \eta \ll 1, g_R = 0$
- Wall, Neumann : no flux $\alpha_R = 0, g_R = 0$

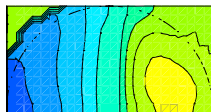
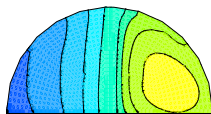


► Primary/secondary fluid exchanged power :

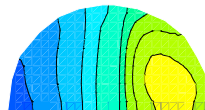
Cells	Exchanged power (MW)	Relative error
143,360 (body-fitted)	1.48873	-
7,200	1.49054	1.1×10^{-3}
57,600	1.48851	1.5×10^{-4}
460,800	1.48763	7.4×10^{-4}

► Specific enthalpy iso-values :

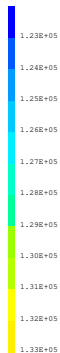
57,600 cells



restriction →



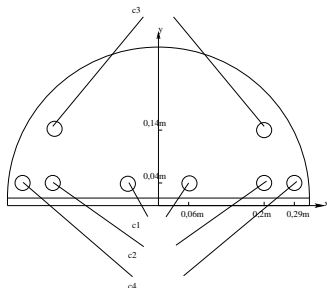
VAL - J80
> 1.18E+05
< 1.40E+05



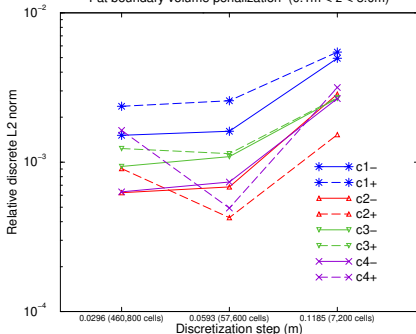
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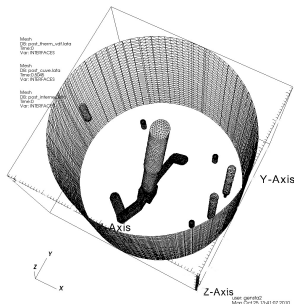
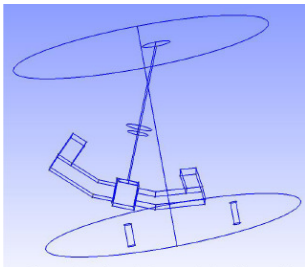
► Mesh convergence (L^2 norm) :



Clotaire BM

Fat boundary volume penalization ($0.1m < z < 8.0m$)

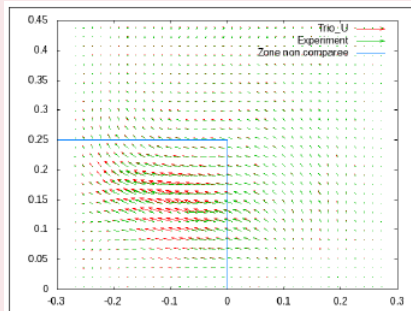
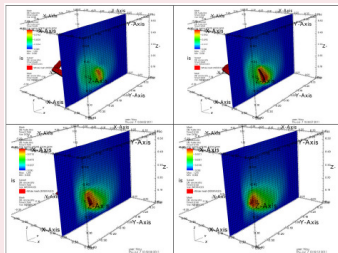
- ▶ Hydraulic validations of flows induced by a stirrer
- ▶ Thermal-hydraulic validations of flows around a rod
- ▶ Cold crucible full-scale simulation involving bubbling



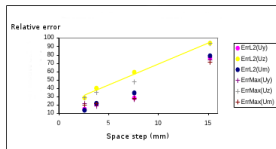
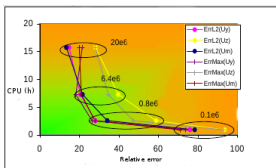
- ▶ IBC Lagrangian meshes from CAD
- ▶ Regular Eulerian meshes

Hydraulic validations

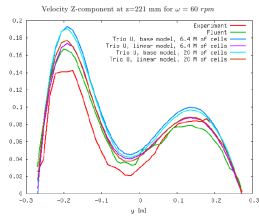
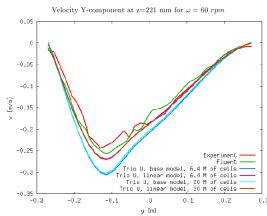
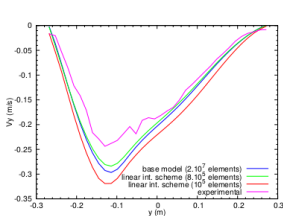
- ▶ Experimental device filled with viscous oil ($Re \approx 100$) - PIV -
- ▶ Fluent body-fitted sliding-mesh simulations (1 M cells with AMR)
- ▶ Velocity-profile comparisons on a cut-plane and lines



- **Base model** : 0.05 → 20 M cells on 2 to 400 CPUs : rate of convergence about **0.6**



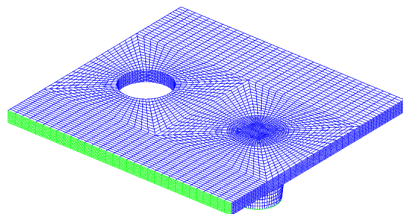
- **Linear** vs base model : cost ↘ for a given accuracy (**1 M cells is quasi-optimal**) : rate of convergence about **1.0**



- Trio_U results compare well with experiment and Fluent ones

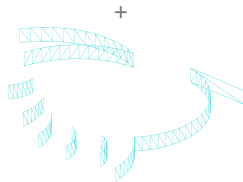
Flow in a hydraulic diode

- ▶ Completely 3D case
- ▶ Water (tabulated EOS)
- ▶ Pressure of 50 bar
- ▶ inlet flow of $4.680 \text{ kg}\cdot\text{s}^{-1}$

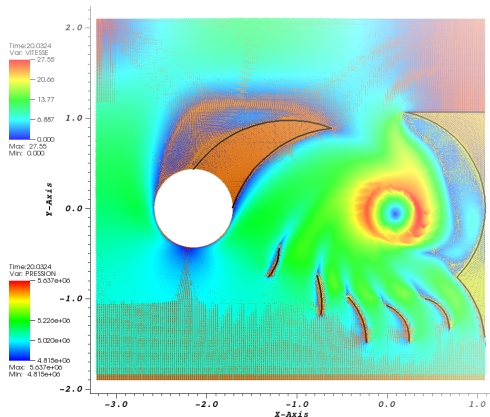


Studies

- ▶ Mesh convergence
- ↔ *Pressure drop coefficient*
- ↔ *Finest grid = reference*



Example of obtained pressure and velocity maps



Vectors

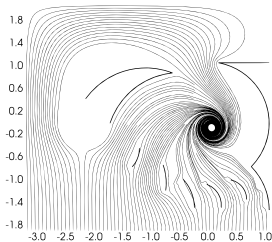
Velocity in m.s^{-1}

Colors

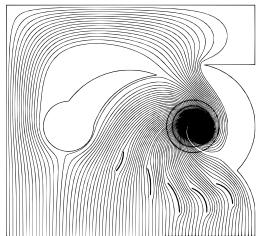
Pressure in bar

Analysis

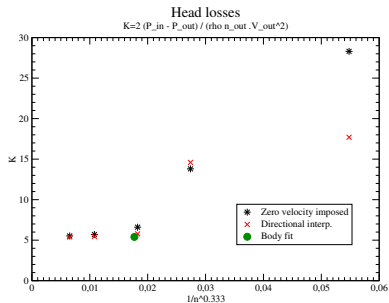
- ▶ Expected behavior
- ▶ Consistent with [Bel18]



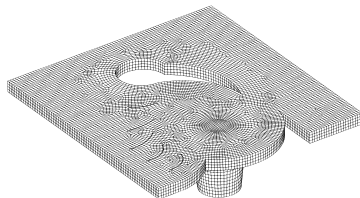
PDF



« Body-fitted »

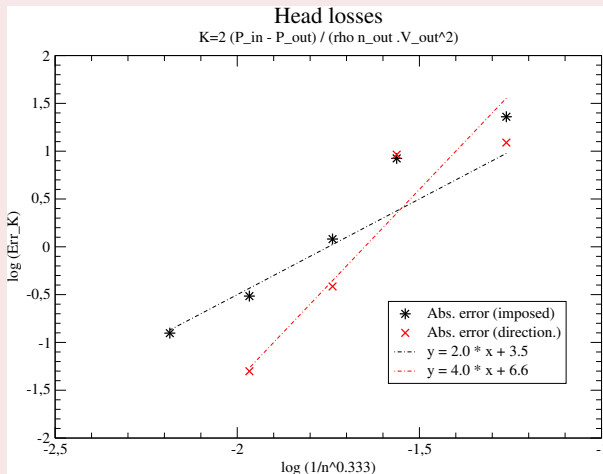


Head loss



« Body-fitted » mesh

Mesh convergence



↪ Convergence **enhanced by the interpolation** of the imposed velocity

Conclusions and perspectives

Conclusions

- ▶ 20 years of R&D in fictitious domain methods for nuclear-industry applications
 - ISI, JEBC, DF and PDF methods for thermal-hydraulic solvers
 - FE, FD and FV discretizations
 - Various linear interpolations near the immersed boundary
 - Validation on academic test cases
- ▶ Industrial applications : IFS, waste vitrification process, hydraulic diodes

Conclusions

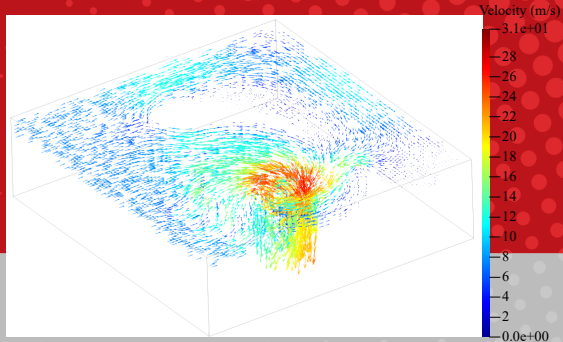
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- ▶ Industrial applications : IFS, waste vitrification process, hydraulic diodes

Perspectives

- ▶ Immersed turbulent wall laws
- ▶ Shape optimization during computations
- ▶ AMR

Thank you for listening ...

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